Modelling of Multi-Parameter-Level Simulation Data to Create an Enhanced Cordier Diagram for Radial Turbocompressors

iSimT-24 Symposium on Innovative Simulations in Turbomachinery Yannick Lattner, M.Eng

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- 1. Introduction
- 2. Computational Model
- 3. Nested Design of Experiments
- 4. Surrogate Modeling
- 5. Applications

Parameterization, simulation and data-modeling as an integral part of the PhD-thesis:

A Multi-Level Design Space-Based Enhanced Cordier Diagram for Radial Turbocompressors

(Currently being reviewed at Bauhaus-Universität Weimar)



Concept of the enhanced Cordier diagram

- Shall provide combinations of specific speed σ and specific diameter δ using the Cordier line
- Iterations of rotational velocity or diameter to obtain feasible designs, change the location in the Cordier diagram
- Cordier line is not a functional, precise approach for modern radial turbocompressor designs



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Concept of the enhanced Cordier diagram

The Cordier diagram

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Impeller exit relative blade angle distribution

Concept of the enhanced Cordier diagram

- Fails to provide precise combinations of rotational velocity and impeller diameter for modern machines
- Does not provide any indication regarding a future compressor's efficiency
- Neglects significant parameter influences



Concept of the enhanced Cordier diagram

A digital, enhanced Cordier Diagram

- Re-enable the indented use
- Provide addition data to weight possible designs
- Operate as a digital tool with application-based outputs
- Based on simulation data of modern radial turbocompressors
- Inputs must be available at the start of the design process

Requirements to the computational model

- Complex parameter space
 - Differentiation of influences introduced by geometry design space and machine design space
 - Machine design process
 - Parameterized CAD Geometry
 - CFD-based speed line computation
 - Structural simulation (FEA)
- Automated workflow
- Compatible parameter spaces

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Machine Design Model

- Inputs:
 - Ambient conditions
 - Outflow pressure
 - Ambient volume flow
 - Axial impeller extent ratio
 - Impeller exit relative blade angle
 - Circumferential blade extent
 - Number of blades
 - ...

• Outputs:

- All principal dimensions
 - Diameters
 - Blade angles
 - Thermodynamic properties
 - Chord Reynolds number
 - ...

Geometry Design

- Based on non-dimensional parameters (Angles / ratio to D₂)
- Including parameterized impeller disc
- Reduced parameter set controls multiple design parameters by global parameters



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Geometry Design

- Reduced parameter set controls multiple design parameters by global parameters
- Allows for significant reduction of sample size with neglectable reduction of result parameter range



CFD Model

- CFX Turbomachinery Setup
- Exit-corrected mass flow rate
- Convergence evaluation using CoV
- Averaging approach for unsteady results



Speed Line Computation

- Fully automated speed line computation tool ¹⁾
- Choke point defined at quasiconstant mass flow rate
- Surge at maximum of static outflow pressure
- Direct Peak-Efficiency point identification



¹⁾ "Physics-Based Surge Point Identification for Unsupervised CFD-Computation of Centrifugal Compressor Speed Lines" in Energy Conversion and Management: X, Yannick Lattner, Marius Geller and Michael Kutz, DOI: <u>10.1016/j.ecmx.2022.100337</u>

Neural network-based speed Line interpolation



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FEA Model

- Load due to rotational velocity
- No thermal or dynamic effects
- Evaluation of:
 - Maximum deformation and von-Mises stress
 - Leading and trailing edge deformations
 - Elastic strain energy (density) for multiple partitions



Hybrid Surrogate modeling

- Flexible and universal approach to surrogate modeling using multiple component surrogate models (CSMs)
- Each component surrogate model is individually trained
- Hybrid surrogate model is composed by all by optimizing individual weights
- In this approach, combination of:
 - Polynomial regression
 - Neural networks
 - Kriging

Hybrid Surrogate modeling

- Scanning-test-set cross validation
- Measure for surrogate modeling
 - WR2: Weighted coefficient of determination:

$$WR2 = 1 - \frac{SSE_{\text{Test}} + \alpha SSE_{\text{Training}}}{SST_{\text{Test}} + \alpha SST_{\text{Training}}}$$

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- Separation of machine design space and geometry design space
- First level: machine design of experiments
- Second level: geometry designs of experiments for each machine design



Machine design of experiments

• 50 Machine designs using LHS design

Parameter	\mathbf{Symbol}	DOE Properties
Sampled propertie	es	Range
Cordier line position	$s_{ m Cordier}$	0 to 1
Outflow pressure	$p_{ m out}$	$2 \times 10^5 \mathrm{Pa}$ to $4 \times 10^5 \mathrm{Pa}$
Ambient volume flow rate	$\dot{V}_{ m amb}$	$0.5{ m m}^3{ m s}^{-1}$ to $4{ m m}^3{ m s}^{-1}$
Axial impeller extent ratio	$ u_L$	0.2 to 0.4
Trailing edge blade angle	eta_2	40° to 75°
ircumferential blade extent	$ heta_2$	35° to 55°
Constant properti	ies	Value
Ambient pressure	$p_{ m amb}$	$1 \times 10^5 \mathrm{Pa}$
Ambient temperature	$T_{ m amb}$	$288.15\mathrm{K}$
Iub to shaft diameter ratio	$D_{1_{ m hub}}/D_{ m shaft}$	1.25
Blade thickness ratio	$ u_s$	0.01
Number of blades	\boldsymbol{z}	13



Geometry design of experiments

• 50 geometry designs using LHS design for each machine design point

Parameter	\mathbf{Symbol}	DOE Properties
Sampled properties		Range
Hub control points in meridional direction	Hub_Mer	-1 to 1
Hub control points in spanwise direction	Hub_Span	0 to 1
Shroud control points in meridional direction	Shroud_Mer	-1 to 1
Shroud control points in spanwise direction	Shroud_Span	0 to 1
Leading edge hub position	LE_Hub_Pos	0.05 to 0.15
Leading edge offset angle hub to shroud	LE_Shroud_Offset	0° to 10°
Blade twist (Leading Edge)	Twist	-10° to 10°
Blade rake (Trailing Edge)	Rake	-25° to 25°
Intensity of blade's S -shape (hub)	Theta_Hub_S	0 to 1
Intensity of blade's S-shape (shroud)	Theta_Shroud_S	0 to 1

Solving

- CFD
 - 2481 / 2500 successfully computed speed lines
 - More than 19 TB of data
 - More than 35.000 CFD simulations
 - Computational time around 6 months
- FEA
 - All 2500 designs were successfully computed
 - 2.7 TB data

Initial result evaluation



Initial result evaluation



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Speed line interpolation

- Modeled as function of exitcorrected mass flow rate:
 - Mass flow rate
 - Total-to-static pressure ratio 🚆
 - Polytropic efficiency
- Additionally, spline fitting was conducted



Speed line interpolation

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Geometry design of experiments

- Inputs: Geometry design parameters
- Outputs:

Parameter	Symbol	Unit
Speed line parameters		
Peak efficiency	$\eta_{ m pol_{max}}$	_
Peak efficiency mass flow rate	$\dot{m}_{\eta-\mathrm{max}}$	${\rm kgs^{-1}}$
Peak efficiency total-to-static pressure ratio	$\Pi_{\mathrm{t-s}_{\eta-\mathrm{max}}}$	_
Choke efficiency	$\eta_{ m pol_{choke}}$	_
Choke mass flow rate	$\dot{m}_{ m choke}$	${\rm kgs^{-1}}$
Choke total-to-static pressure ratio	$\Pi_{\rm t-s_{\rm choke}}$	_
Surge efficiency	$\eta_{ m pol_{surge}}$	_
Surge mass flow rate	$\dot{m}_{ m surge}$	${\rm kgs^{-1}}$
Surge total-to-static pressure ratio	$\Pi_{t-s_{\rm surge}}$	_
Normalized speed line width	$\Delta \dot{m}_{ m norm}$	_
Total-to-static pressure ratio spline parameter (\dot{m} -coordinate)	$\chi_{\mathtt{TSPR}_{\dot{m}}}$	_
Total-to-static pressure ratio spline parameter (Π_{t-s} -coordinate)	$\chi_{\mathtt{TSPR}_{TS^{'}PR}}$	_
Polytropic efficiency spline parameter (\dot{m} -coordinate)	$\chi_{{\tt Eff}_{\dot{m}}}$	_
Polytropic efficiency spline parameter ($\eta_{\rm pol}$ -coordinate)	$\chi_{{\tt Eff}_{\dot\eta}}$	_

Structural load parameters		
Maximum value of the von Mises stress		Pa
Maximum value of the deformation		m
Directional deformation of the leading edge at the shroud, collinear		m
Directional deformation of the trailing edge at the shroud, collinear		m
Elastic strain energy of the full impeller model	U_{impeller}	J
Elastic strain energy of the blade region	U_{blade}	J
Elastic strain energy of the disc region		J
Elastic strain density of the full impeller model		${ m J}{ m m}^{-3}$
Elastic strain density of the blade region	$u_{ m blade}$	${ m J}{ m m}^{-3}$
Elastic strain density of the disc region	$u_{ m disc}$	${ m J}{ m m}^{-3}$
Volume of the full impeller model	V_{impeller}	m^3
Volume of the blade region	$V_{ m blade}$	m^3
Volume of the disc region	$V_{ m disc}$	m^3

Geometry design of experiments

- Hybrid surrogate model is trained 50 times (each machine design point) for each output parameter = 1350 surrogate models
- Mean WR2 = 0.9483

Machine design point-wide optimization

- Optimization of geometry design parameters for each machine design point:
 - Achievable efficiency and achievable speed line width





Machine design of experiments

Inputs:

Sampled:

- Ambient volume flow rate
- Total-to-static pressure ratio
- Axial impeller extent ratio
- Circumferential blade extent
- Impeller exit relative blade angle Derived:
- Specific diameter
- Specific speed
- Flow coefficient
- Polytropic work coefficient
- Chord Reynolds number

- Outputs:
 - Achievable Efficiency
 - Achievable Speed Line Width
 - Associated speed line parameters
 - Associated geometry parameters
 - Machine design point-wide structural result parameters
- WR2 for main parameters > 0.99

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RESULTS

Duty-specific Cordier lines



RESULTS

Direct optimization, initial guess for impeller geometry



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RESULTS

Efficiency correlations





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THANK YOU

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